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Experimental Guidelines for the Design of Turbine Rotor Fragment Containment Rings



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Naval Air Propulsion Center Trenton, New Jersey

July 1988

FINAL

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PREFACE

This report has been prepared by the Naval Air Propulsion Center (NAPC), Trenton, New Jersey, under National Aeronautics and Space Administration (NASA) Defense Purchase Request C41581-B Modification No. 10 from the Lewis Research Center, NASA, Cleveland, Ohio 44135, and by direction of the Federal Aviation Administration. Mr. Robert Johns and Dr. Chris Chamis of the Lewis Research Center served as program monitors. Their contributions and help during this program are greatly appreciated. The authors would like to thank the Boeing Military Airplane Development Organization, a part of the Boeing Aerospace Company, for conceptual and analytical support in the fabrication of the cloth containment rings.

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LIST OF ILLUSTRATIONS

Figure		Page
1	Test Setup for Task l	5
2	J65 Turbine Rotor Characteristics	6
3	J65 Rotor Modified	7
4	Kevlar-Nylon Tests	8
5	NAPC Wrapping Process	9
6	Kevlar Containment Ring	10
7	Nylon Containment Ring	11
8	Containment Ring Characteristics	12
9	Ballistic Nylon Uncontained Failure	13
10	Kevlar 29 Uncontained Failure	14
1:	Containment Ring Design	15
12	Single Blade Modification	16
13	Triple Blade Modification	17
14	Test Scheme for Task 2	18
15	Test Matrix for Task 2	18
16	Fragment Energy Versus Containment Ring Thickness	19
	LIST OF TABLES	
Table		Page
1	Rotor Fragment Containment Experiments	20

Blade Containment Experiments

2

21

TABLE OF CONTENTS

	Page
EXECUTIVE SUMMARY	vii
INTRODUCTION	1
METHOD OF TEST	1
DESCRIPTION OF TEST AND DISCUSSION	2
CONCLUSIONS	4
REFERENCES	22
APPENDICES	

- ${\tt A}$ Cloth Containment Rings Manufacturing Techniques ${\tt B}$ Distribution List

EXECUTIVE SUMMARY

The experimental program supporting parametric studies of rotor fragment containment was developed and conducted by the Naval Air Propulsion Center (NAPC) under National Aeronautics and Space Administration (NASA) sponsorship. The formal reporting of this program was sponsored by the Federal Aviation Administration Technical Center (FAA) because of the relevance of the information to propulsion safety. The program objective was to develop guidelines for the design of optimum weight containment rings for gas turbine engine rotor fragments.

PARTICIPATION PROFESSORS

The test program being reported consisted of two tasks. Task 1 was an investigation of the containment of rotor sectors by cloth rings. Task 2 determined the engine casing thickness required for single and triple blade containment.

One phase of Task I was an investigation of the capabilities of Kevlar fabric as a lightweight containment device as opposed to 4130 steel which was reported in reference 2. Another phase of Task I was to compare the containment performance of rings manufactured from ballistic nylon fabric to rings manufactured from Kevlar 29 fabric. Task I consisted of a series of rotor fragment containment experiments in which rotors were modified to fail at their respective design speeds into three, pie-shaped fragments. These fragments impacted rings made from aluminum mandrels wrapped with either Kevlar 29 fabric or ballistic nylon fabric. The results of these rotor containment experiments indicated that Kevlar is superior to nylon but still considered inadequate for containing rotor fragments.

The objective of Task 2 was to determine the minimum engine casing thickness required to contain single and triple blade failures. This task consisted of a series of blade containment experiments in which Pratt & Whitney JT3D and JT8D turbine blades were modified to fail at 100 percent design speed and generate single and triple blade fragments. These fragments impacted steel rings whose radial thickness was varied until fragment(s) containment was achieved. The results of these experiments indicated that the minimum engine casing thickness required to contain a single JT8D blade is 0.187 inch and to contain a single JT3D blade is 0.250 inch. The test results showed that a minimum casing thickness of 0.375 inch is required to contain either a JT3D or JT8D triple blade failure. The results attained in Task 2 can be used as a model for containment devices used in any military or commercial aircraft having engines comparable in size to the JT3D or JT8D.

INTRODUCTION

The Rotor Fragment Protection Program was sponsored by National Aeronautics and Space Administration (NASA) and conducted by Naval Air Propulsion Center (NAPC). Formal reporting of the program was sponsored by the Federal Aviation Administration (FAA) Technical Center because of the relevance of the information to propulsion safety. The objective of the program was to develop guidelines for the design of lightweight devices that will be used on aircraft to protect passengers and the aircraft structure from lethal and devasting fragments generated by failure of gas turbine engine rotating components.

Previous reports published by NAPC in conjunction with NASA which document the progress of this program are listed as references 1 through 7.

This report presents the results of a parametric test program that was conducted to provide guidelines for the design of gas turbine rotor fragment containment rings.

METHOD OF TEST

Task 1. Test results presented in this report were obtained using basically the same equipment and techniques described in references 4 and 5. Figure 1 shows a typical test installation. The test procedure was as follows. The test rotor, modified to fail and produce particular shaped fragments at a specified speed, is connected to the air-drive turbine by an arbor. The containment ring under test is freely suspended by guide wires and is concentrically positioned around the test rotor. The axial, midsection of the ring is positioned to coincide with the test rotor's plane of rotation, and radial tip clearance between the rotor and ring is maintained at 0.50 inch (1.27 centimeters). The spin chamber is evacuated to 10 millimeters (mm) Hg pressure in order to reduce the aerodynamic drag on the test rotor and thus reduce the power required to accelerate the rotor to its failure speed. In order to record the fragment release speed, an impacttriggering strip is fixed to the inner diameter of the containment ring. the fragment is released and makes contact with the inner surface of the ring, the low voltage signal of the triggering strip is interrupted, indicating the time of failure which is correlated to the speed at that time. The containment criterion for the turbine rotor fragment containment tests of Task 1 was as follows: The cloth ring is an acceptable turbine rotor fragment containment system if the fragments do not penetrate the outer layer of cloth.

Task 2. The blade containment experiments performed in Task 2 were conducted in a containment chamber that is 4 feet in diameter and 1 foot high. The test rotor, modified to release single or triple blade fragments, is connected to the air-drive turbine by an arbor. The test containment ring is placed at the bottom of the containment chamber and is concentrically positioned around the test rotor. The axial midsection of the ring is positioned to coincide with the test rotor's plane of rotation. The containment chamber is evacuated to 0.5 mm Hg pressure in order to reduce the aerodynamic drag on the test rotor and thus the power required to accelerate the rotor to its designed 100 percent speed. In order to record the blade release speed, an accelerometer, which detects vibrations in the chamber, is affixed to the chamber lid. The shock that is produced by the blade failure is detected by the accelerometer indicating the time of failure which is correlated to the speed of that time.

The containment criteria for the turbine blade containment tests of Task 2 were as follows: The engine casing is acceptable for blade containment if the damage from rotor blade failures is contained by the engine case; i.e., without causing significant rupture or hazardous distortion of the engine casing and/or the expulsion of blades through or beyond the perimeter of the engine case or shield.

DESCRIPTION OF TEST AND DISCUSSION

Task 1. Table 1 lists the rotor fragment containment experiments that were conducted to investigate containment capabilities of cloth rings. It also describes the materials used and the conditions of each experiment.

Four containment experiments were conducted to investigate the performance of Kevlar® 29 fabric as a lightweight containment device as opposed to 4130 steel. In three of these experiments, the General Electric T58 engine power turbine rotors were modified to fail at design operating speed into three equal pieshaped fragments. In the fourth containment experiment, a General Electric T58 engine power turbine rotor was modified to fail at 100 percent design speed into six equal pie-shaped fragments. These experiments aided in the development of the final design configuration of cloth containment rings which was then used in the rotor fragment containment evaluations comparing the containment performance of rings manufactured with ballistic nylon to those manufactured with Kevlar 29. Eight of these nylon versus Kevlar rotor fragment containment tests were conducted using Curtiss-Wright J65 turbine rotors (figure 2) modified to fail at design speed into three equal pie-shaped fragments (figure 3).

The fragments from these rotor containment tests impacted containment rings made from lightweight aluminum drums, 1/32 inch thick, wrapped with either Kevlar 29 or ballistic nylon fabric. The material was folded on the bias and stitched with one, two, and three rows of Kevlar or nylon thread to give the best energy absorption capability. Figures A-l through A-5 of the appendix define the Kevlar and nylon containment ring manufacturing technique. This technique was developed by the Boeing Company, who fabricated and wrapped the Kevlar containment rings used in the first eight rotor fragment tests (see figure 4). In the last four containment tests (three ballistic nylon tests and one Kevlar 29 test), the containment rings were manufactured by the Pioneer Parachute Company and wrapped by NAPC personnel. An example of the NAPC wrapping process is shown in figure 5. Once wrapped, the cloth containment rings can be seen in figures 6 and 7. A brief summany of the cloth containment ring characteristics is presented in figure 8.

Results of the rotor fragment containment experiments conducted in Task 1 are shown in table 1. In the Kevlar 29 evaluation experiments, only one of four General Electric T58 turbine rotors was contained. In the kevlar 29 versus ballistic nylon experiments, only two Kevlar 29 rings contained the fragments generated by a Curtiss-Wright J65 turbine failure and none of the ballistic nylon rings contained the fragments. Figures 9 and 10 are examples of the nylon and Kevlar fabric damage after uncontained failures. The results of these cloth containment ring tests indicated that Kevlar 29 is superior to ballistic nylon, but both are considered inadequate for rotor containment.

Task 2. Table 2 lists the blade containment experiments that were conducted to determine the minimum engine casing thickness required for single and triple blade containment. This table also describes the materials used and the conditions of each experiment.

To determine the minimum engine casing thickness, a series of blade containment experiments was conducted. In these containment experiments, blades from fully bladed JT3D (second stage) and JT8D (first stage) low pressure turbine rotors were modified to fail at 100 percent design speed (6850 and 8800 revolutions per minute (rpm) respectively) and impact containment rings of various thickness (figure 11) made from A-286 steel. (JT3D and JT8D rotors were chosen as representative of large turbine rotors in use today.) The blade containment experiment results reported herein were generated by single and triple blade releases. Single blade release is accomplished by appropriate notching of a blade causing it to fail at the rotor's 100 percent design speed. Figure 12 shows a single blade notch configuration. Triple blade release is accomplished by modifying the rotor, as shown in figure 13, causing three blades to separate from the rotor at 100 percent design speed. Soveral preliminary tests were conducted, using 304 stainless steel containment rings, to verify the appropriate blade notching configuration to be used in achieving blade release at design speed.

The actual blade containment experiments used containment rings made from A-286 steel with an axial length of 9 inches. The radial thickness of the containment rings was varied from 0.140 inch to 0.387 inch. To optimize testing and reduce cost, a test scheme (figure 14) was developed. This test scheme was designed as follows: An arbitrary ring thickness was chosen, based on engineering judgement, as the starting point, and a containment test was conducted. If containment was achieved, the containment ring thickness would be reduced and another blade containment test would be conducted. If containment was not achieved, the containment ring thickness would be increased for the next containment test. This process was continued until containment was achieved. The results of these blade containment experiments are presented in matrix form in figure 15. These results indicate that the minimum engine casing thickness required to contain a single JT8D blade is 0.187 inch and to contain a single JT3D blade is 0.250 inch. The minimum engine casing thickness required to contain three blades from either a JT3D or JT8D rotor is 0.375 inch.

Using this test data, a simplified formula has been devised whereby the thickness of the containment ring required to contain fragments of specified energy may be estimated. Since the blade containment tests were conducted with containment rings made from A-286 steel, the formula is valid only for this ring material. It is expected that variations in ring material can be expressed by changes in the formula constants. The formula is

 $t = .0014E \cdot 48$

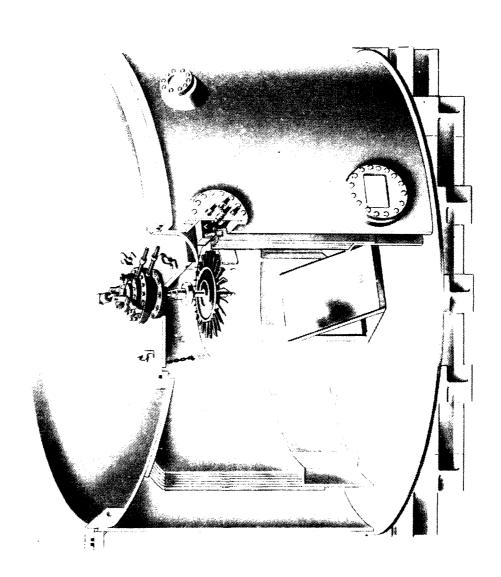
where t = thickness of the A-286 steel containment ring, inches

E = kinetic energy of the fragment, in-1bs.

Figure 16 shows the test data points plotted as fragment energy versus containment ring thickness. The solid line in figure 16 is a plot generated from the formula above. It is being used as a conservative approximation of the minimum thickness required for an $\Lambda-286$ steel containment ring to be successful at the corresponding fragment energy. However, due to the limited number of test specimens used in developing this formula, exact containment ring thickness should not be derived from this relationship.

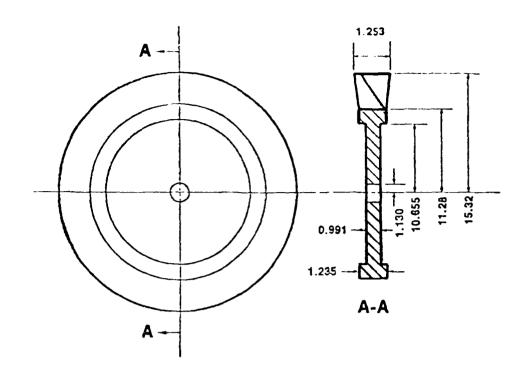
CONCLUSIONS

- Task 1. Neither Kevlar [®] 29 nor ballistic nylon fabrics are adequate for rotor containment. However, Kevlar 29 did perform better than ballistic nylon in these tests.
- $\frac{Task\ 2}{Dlade}$. The minimum engine casing thickness required to contain a JT8D single blade failure is 0.187 inch and to contain a JT3D single blade failure is 0.25 inch. Both the JT8D and JT3D experiments required a minimum casing thickness of 0.375 inch for triple blade containment. Since the JT8D and JT3D turbine rotors are representative of most large turbine engine designs used in commercial aircraft, the results of this testing can be generalized for any turbine engine having blade fragment energies comparable to the JT8D and JT3D turbine blades.



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FIGURE 1. TEST SETUP FOR TASK 1



TYPE ROTOR: J-65 SECOND STAGE TURBINE

(MODIFIED, UNSLOTTED)

ROTOR WEIGHT: 128 LBS (APPROX.)

ROTOR INERTIA: 9410 LB-IN2 (NOMINAL)

	<u>DISK</u>	BLADES
MATERIAL :	17-22A FERRITIC STEELS	INCONEL 700
PROPERTIES :		
SU	142K PSI	148.5K PSI
SY	128.5K PSI	192K PSI
EU	18%	18%
HD	311 BHN	311 BHN

FIGURE 2. J65 TURBINE ROTOR CHARACTERISTICS

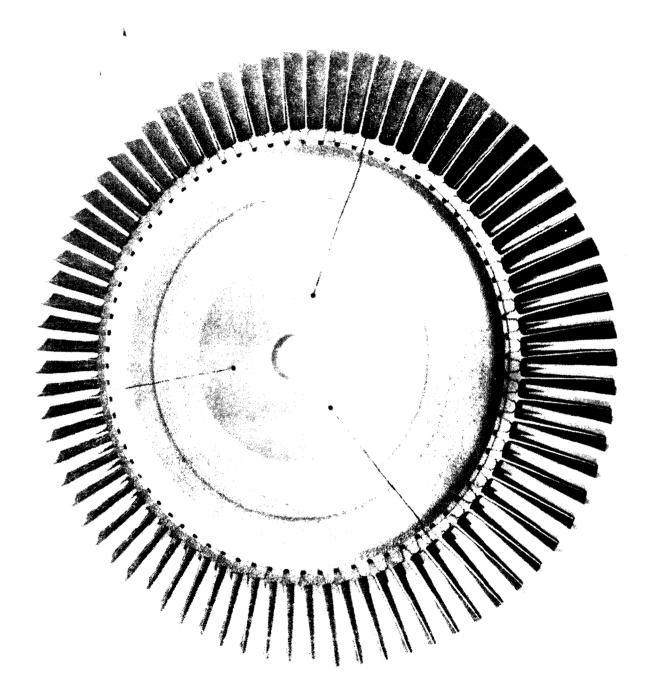


FIGURE 3. J65 ROTOR MODIFIED

T58 Rotors

Kevlar - Boeing - Four (4) Tests

215 - 06/14/76 - Three (3) Fragments - Partial Containment è. 1:

218 - 09/02/76 - Three (3) Fragments - Contained No.

219 - 10/19/76 - Six (6) Fragments - Not Containment Š

220 - 12/07/76 - Three (3) Fragments - Partial Containment Š. ъ. 4

J65 Rotors

Kevlar - Boeing - Four (4) Tests

221 - 02/16/77 - Three (3) Fragments - Contained No.

Fragments - Contained 222 - 04/27/77 - Three (3) Š Š

226 - 09/01/77 - Three (3) Fragments - Not Contained 229 - 11/07/77 - Three (3) Fragments - Not Contained . ov Š.

Nylon - Pioneer Parachute Company - Three (3) Tests

Fragments - Not Contained ව 241 - 12/12/80 - Three Š. 7:

No. 244 - 02/02/81 - Three (3) Fragments - Not Contained - Not Contained Fragments - Three (3) No. 243 - 01/30/81

Kevlar - Pioneer Parachute Company - One (1) Test

1. No. 245 - 02/04/81 - Three (3) Fragments - Not Contained

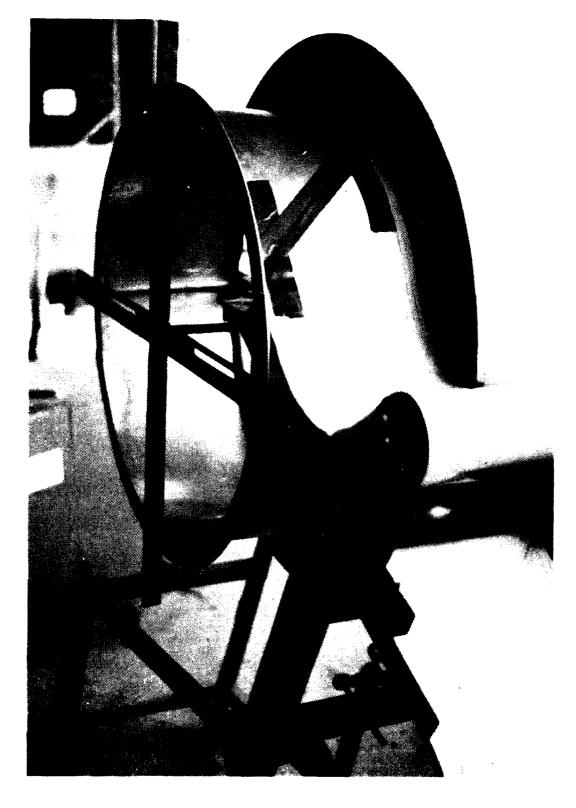
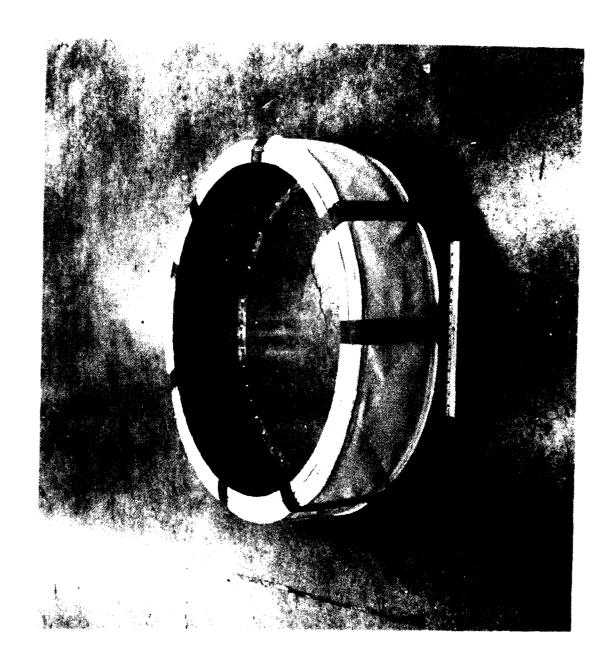
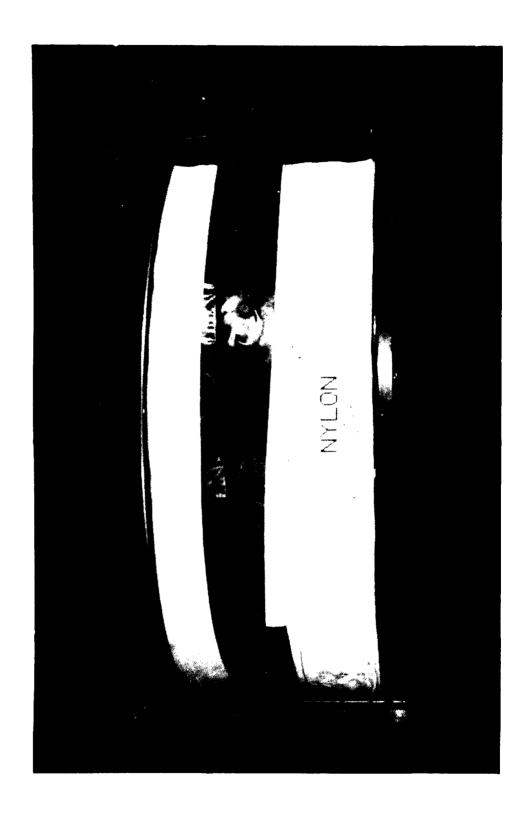
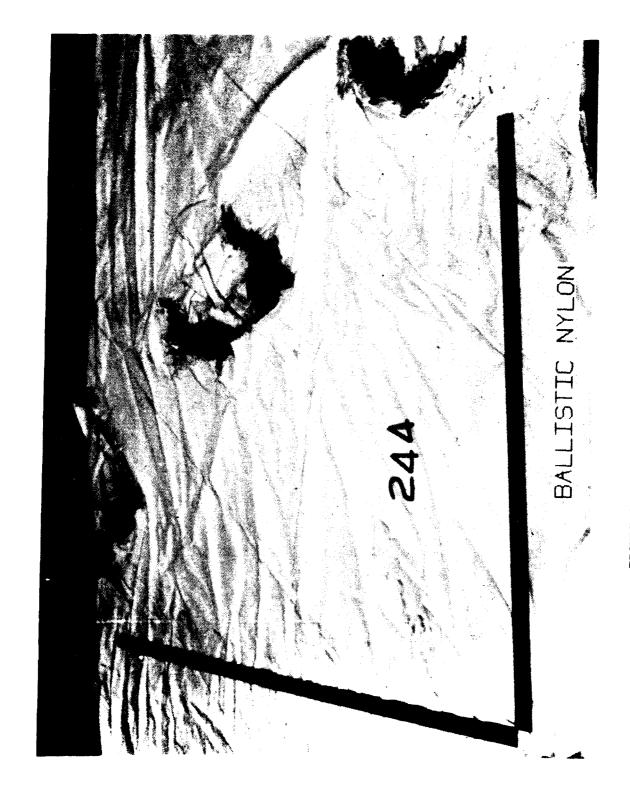


FIGURE 5. NAPC WRAPPING PROCESS





MATERIAL	BALLISTIC NYLON	KEVLAR 29
QI	32"	32"
AXIAL LENGTH	12.5"	9"-12"
WEIGHT	65 LBS	55-71 LBS



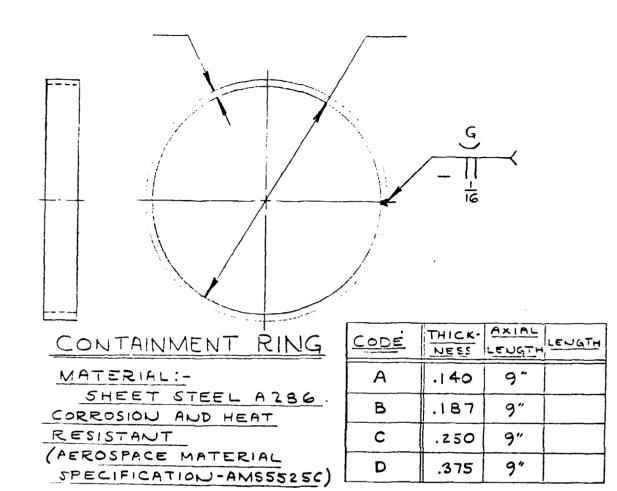
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FIGURE 9. BALLISTIC NYLON UNCONTAINED FAILURE



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FIGURE 10. KEVLAR 29 UNCONTAINED FAILURE



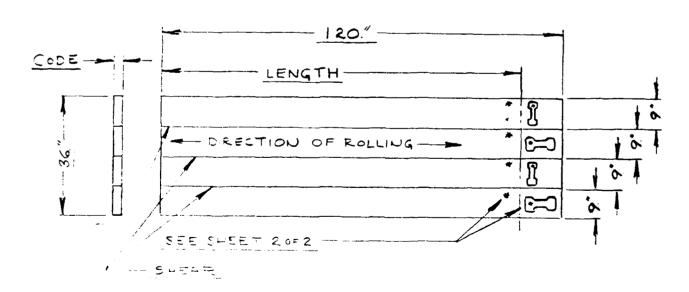
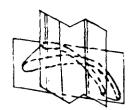


FIGURE 11. CONTAINMENT RING DESIGN

NOTE: MODIFY BLADE



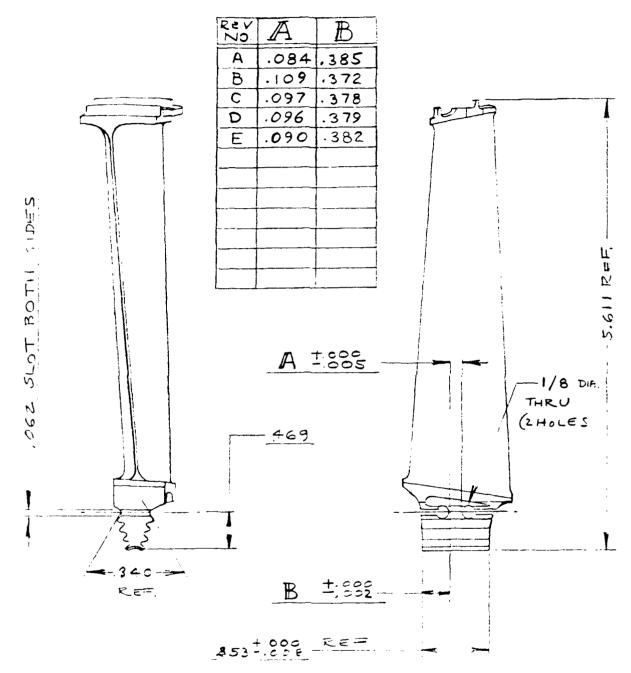


FIGURE 12. SINGLE BLADE MODIFICATION

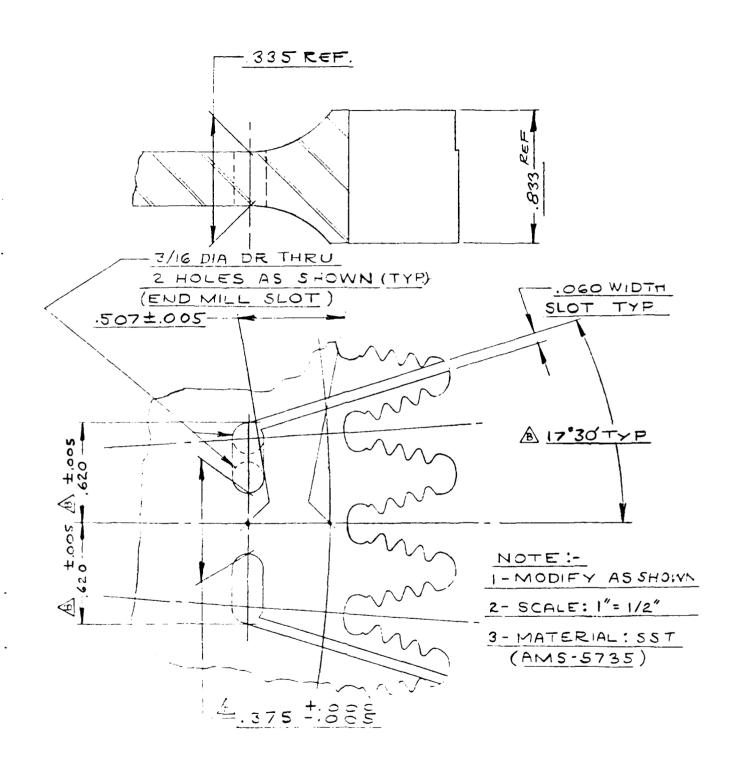


FIGURE 13. TRIPLE BLADE MODIFICATION

FIGURE 14. TEST SCHEME FOR TASK 2

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RING RADIAL THICKNESS

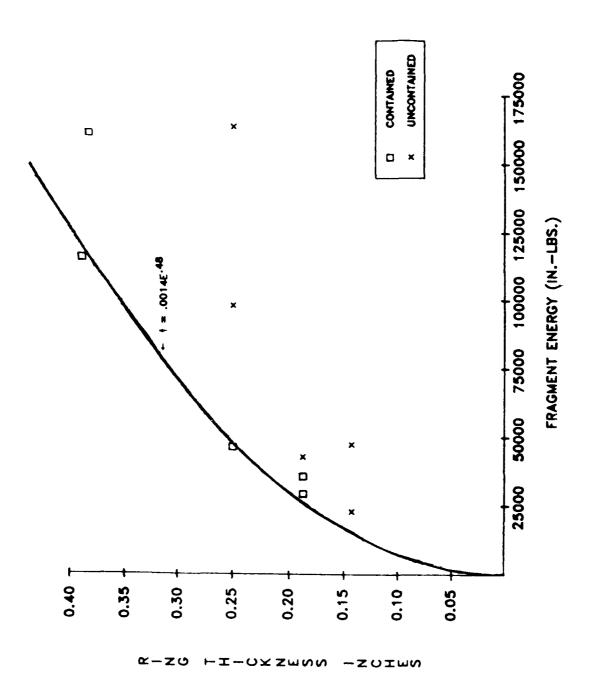


FIGURE 16. FRACMENT ENERGY VERSUS CONTAINMENT RING THICKNESS

ROTOR FRACMENT CONTAINMENT EXPERIMENTS TABLE 1.

2		Æ		POTOR/DISK)19K				C	CONTAINMENT/CONTROL	T/CONTROL			
. 1	-	TEST TYPE	MIBRIAL	DICHES	NO. OF FRACHENTS	Speed Speed	ENERGY CIN-1 BS)	CONFIG-	MATERIAL	ID DOCHES	THILKNESS INCHES	A LOOP	¥£1€H LBS	PESUL 15
÷	215 2147 6	RB PURBINE	982-v		E	19655	95.1828	RING	KEM.AR	15.0	CACL TERMINED	6.0	5 46	PARTIALY (CHIADED
6	22.3	RB TURBINE ROTOR CO.	ΛΕ A-286	Ø+1	m	28549	988709	RING	KEYLAR	15 4375	UACE TERROTED	6.0	6629	CONTAINED
- 2	219 Januara 88	RB NURBINE ROTUR CO	NE A-286	140	9	2016205	995111	BING	KEV AR	15.375	UNDE TETATORED	6.109	5.250	NOT CONTAINED
à	<u>.</u>	PB TURBINE POTOR CO	λ.E. Λ-286	07+1	E	19649	984913	RING	KEV AR	15.375	Cacc Teragoleto	6.125	4 75	PARTIALY
. ₹	221 2/16 m RB		22A FERRITIC P (4) STEEL	38640	E .	HØBE	87.75.788	RING	KEW AR	32.4	25	5 3	57.50	CONTAINED
₹.	222 16.2. 7. 6	AB ROLOR (4)	÷	360.540	· f	22.7B	\$13£.86	PING	KEVLAR S	32.125	275	12.0	7147	CONTAINED
-	226 95.77	RB TURBINE POTOR (4)	Re FERIE	386.540		 ଖଧ୍ୟର	9554916	PING	KEM AR	32.312	25	9	555	NOT CONTAINED
,	229 11.7777 6	RB ROTOR CALL	Real FERRITE	30.649	m	7967	8463482	RING	KEVLAP 3.	62.5	25	12.5	585	NOT CONTAINED
- .	241 JZ 12 H RB	RB ROTOR CO	• 3	30640	æ	B280	₩06/B68	PING	Z X	3225	25	12.5	65	NOT CONTAINED
(₹	243 12-21/91	PB TUMBINE ROTOR (4)	NE FERRITE R (4) STEB.	30.646	T	7734	901353	RING	NYLON	313125	21875	12.5	65	NOT CONTAINED
ן עי	244 2/2/Bit	RB TURBINE ROTOR (4)	÷	30640	Œ	6060	2066€ 6	RING	NALON	37.31.25	21875	571	99	NOT CONTAINED
₹ .	3.00	ROTOR (4)	NE FERRITE R (4) STEB.	38.640	Э	7583	7678243	RING	KEVLAR '60	313125	16.5 VRAPS	125	65	NOT CONTAINE

R TABLES 1 AND 2: DATA COMPILATION FOOTNOTES

- Rotor Burst. (2) Blade Burst. 3
- (3) Turbine Rotor TS8 Engine, Axial Flow Power Turbine Rotor Tip to Hub Ratio = 2.147, Blade Material: SEL, Rotor Material: A-286 (Figures 1 to 6).
- Turbine Rotor Curtiss-Wright J65 Second Stage, Axial Flow.
 Kevlar ring fabricated and wrapped by Boeing Company.
 Kevlar ring fabricated by Pioneer Parachute Company and wrapped by Navy.
- Nylon ring fabricated by Pioneer Parachute Company. 3665
- (8) Turbine Rotor Pratt & Whitney JT3D LPT-2, Axial Flow Turbine, Rotor Tip to Hub Ratio 1.666, Blade Material: AMS 5382, Rotor Material: PWA 1003 (8)
 - (9) Turbine Rotor Pratt & Whitney JT8D LPT-1, Axial Flow Turbine, Rotor Tip to Hub Ratio = 1.458, Blade Material: AMS 5391 (INCONEL 713C), Rotor Material: AMS 5735 (A-286)

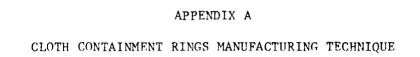
TABLE 2. BLADE CONTAINMENT EXPERIMENTS

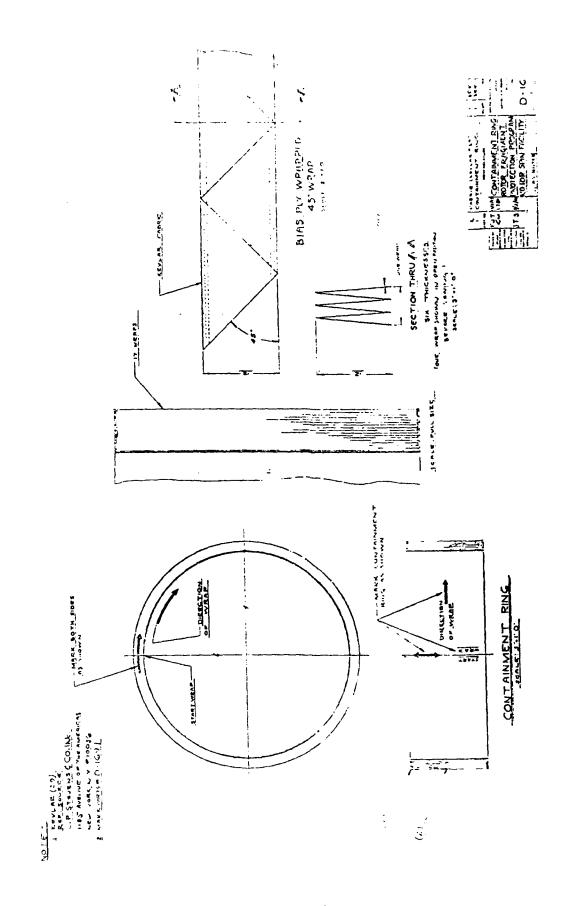
17.7.2. MATCHIAL INTERTION		1,18			ROTOR/015K)19K	•			נסאנ	ADMEN	IT/CONTROL.			
Physical P		-	YPE	MATERIAL		NO. OF FRAGMENTS	SPEED (FFF)	F NEDGY	CONFIDE UPATION	MATERIAL.		THOSE KNESS INCHES	ANDE LENGTH DODES	9971 JHOLET	RESULTS
PM, 1463 3+5975 1 6+56 11319 CAMEDIA ALMINAN 36 1 7 PM, 1683 3+5975 1 6+86 3-5753		ر ' ا	,			1	7152	5,600	FLANCED	ALIMIDAUM		32	7	æ	NOT APPLICABLE
PMA 34-2875 1 6880 39-715 —			JT30	PuA 1883	34 5975	1	6450	41319	FLANGED	ALUNDAUM	9 8	<u>1</u> 32	7	Е	NOT APPLICABLE
PWA 34.5375 1 6380 39419 —	88		JT30	PWA 1003	34.3975	1	6.000	35755	1	1	1		_	-	NOT APPLIEABLE
Physical P	88		JT30 LPT2	PWA 1.003	34.5975	1	6380	39419							NOT APPLIEABLE
Physical P	88	[JT3D LPT2	PWA 1883	34.9375	1	6492	41968							NOT APPLIEABLE
Puls 34.34 1 6145 36275 RING A-286 35.05 140 9 Puls 34.34 1 7020 47397 RING A-286 35.05 140 9 Puls 34.347 1 6736 47397 RING A-286 35.05 140 9 Puls 34.3975 1 6736 45285 RING A-286 35.05 25.0 9 Puls 1,003 3 7245 17490 RING 87/14/ESS 35.0 39 9 A-286 27.35 1 9900 34944 RING 87/14/ESS 20 39 9 A-286 27.35 1 9020 23404 RING 87/14/ESS 20 39 9 A-286 27.35 1 9020 23404 RING 87/14/ESS 20 39 9 A-286 27.35 1 9020 23404 RING	- 69		JT3D LPT2	PWA 1883	34.9975	1	7169	£263.3		304 STAIN ESS STEEL	35.05	390	6	112	NOT APPLIENBLE
Pun	88		JT3D LPT2	PVA 1003	34.94	1	6145	36275	RING	A-286	35.05	781	σ	55	COMIABED
PVA, 1903 34.2875 1 6736 4.9580 RING A-286 35.05 1807 9 1803 34.5875 1 6342 46255 RING A-286 35.05 25.0 9 PAA 34.9375 3 7245 174900 RING SIABLES 35.05 390 99 A-286 27.35 1 9900 3349 RING SIABLES 20 39 9 A-286 27.35 1 9900 3349 RING SIABLES 20 39 9 A-286 27.35 1 9924 2789 RING A-286 28 39 9 A-286 27.35 1 9924 2789 RING A-286 28 39 9 A-286 27.35 1 9924 2789 RING A-286 28 39 9 A-286 27.35 1 9290 2346 RING A-2	88	ſ	JT30 LP12	PVA 1003	34.94	1	7820	47397	RING	A-286	35.05	140	6	98	NOT CONTAINED
Phy Phy	88	l .	JT3D LP12	PWA	34.99.75	1	6736	4.3588	RING	A-286	35.005	781	б	54	NOT CONTAINED
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_	8		JT30 LPT-2	P#A 1603	34.9375	æ	5689	161250	RING	A-286	35.05	T B.	6	STOTI	CONTAINED

• Indicates preliminary test

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FIGURE A-1. KEVLAR CONTAINMENT RING MANUFACTURING DRAWING (1 OF 3)

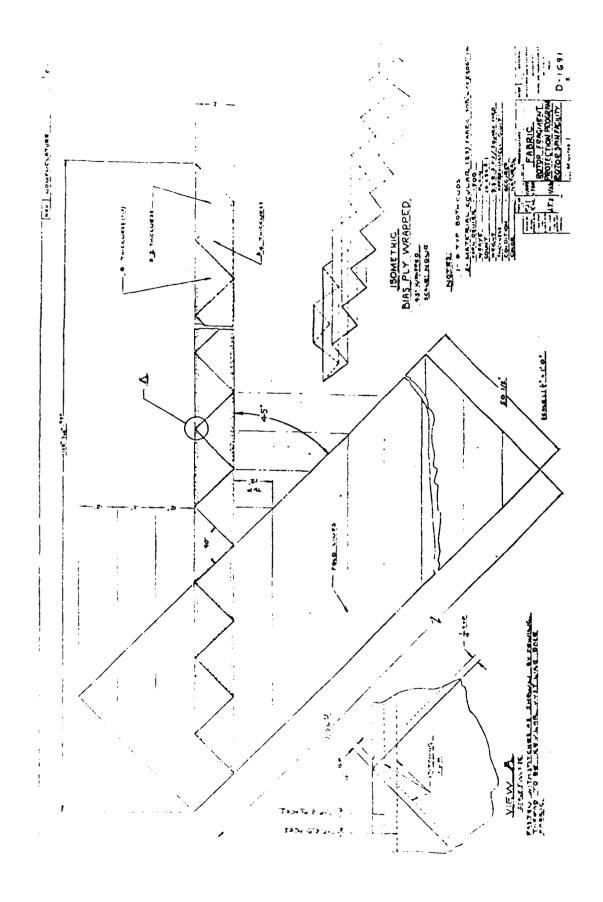


FIGURE A-2. KEVLAR CONTAINMENT RING MANUFACTURING DRAWING (2 0F 3)

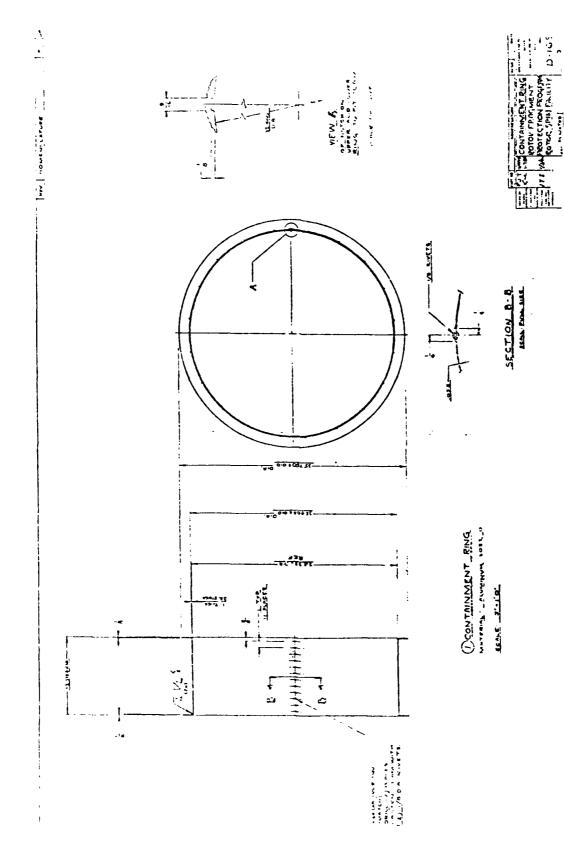


FIGURE A-3. KEVLAR CONTAINMENT RING MANUFACTURING DRAWING (3 OF 3)

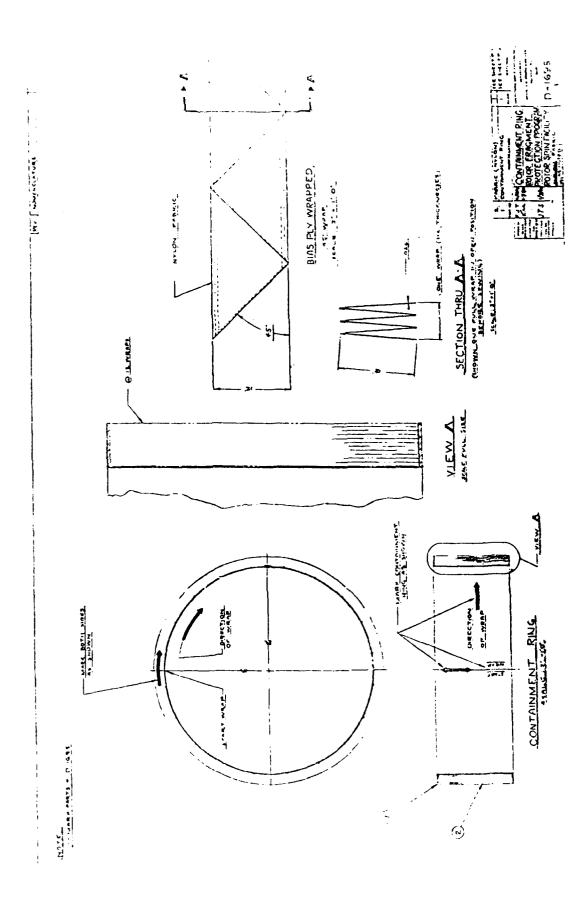


FIGURE A-4. NYLON CONTAINMENT RING MANUFACTURING DRAWING (1 OF 3)

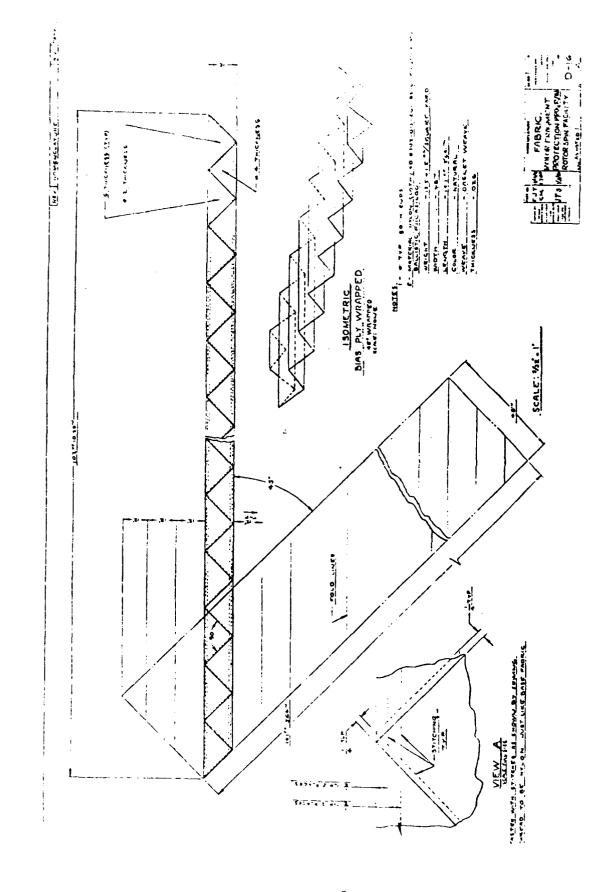
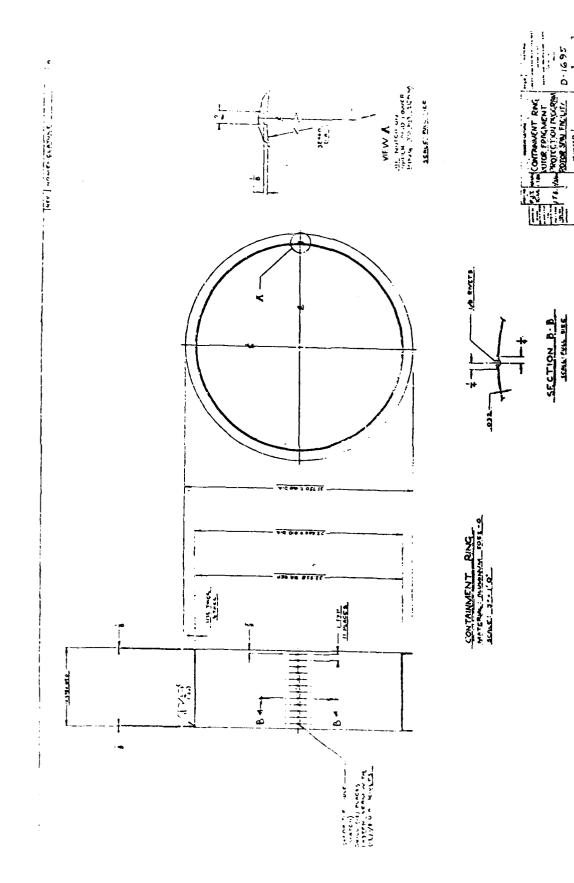


FIGURE A-5. NYLON CONTAINMENT RING MANUFACTURING DRAWING (2 OF 3)



CONTROL OF SECTIONS FOR SECTION OF SECTION O

FIGURE A-6. NYLON CONTAINMENT RING MANUFACTURING DRAWING (3 OF 3)

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